

Artificial compressibility for smoothed particle hydrodynamics using pressure smoothing

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Introduction

Motivation

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- Modelling sloshing fuel in aircraft wings
- Residual acoustic waves in WCSPH interacting with structural model
- Objectives
 - Develop a fully incompressible solution using artificial compressibility (ACSPH) with a focus on FSI solutions
 - 2) Discuss and highlight the links between WCSPH, ISPH and ACSPH (accidental!)
- Success?
 - 1) Yes, ACSPH improved predictions
 - WCSPH (δ-SPH), ISPH and ACSPH are much closer relatives than we thought at the beginning – but perhaps everyone else knew this already?







Putting the Laplacian part on the LHS requires linear solves or pseudo time

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Digestion

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- Incompressible mesh based methods often use PPE; AC less common
- AC meshed methods hardly ever use the Laplacian term (some exceptions). Presumably, this is due to smoother mesh based pressure fields, and the originally proposed ^{Dupuy, JCP 2020} formulation
- AC for SPH has been tried but rarely including Laplacian terms (recently more so, however)
- AC was originally for steady flows, but works for unsteady via a pseudo-time loop, as here

EDAC SPH Ramachandran, CAF 2019

Dual time - Ramachandran, CAF 2021

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Can this be linked to δ -SPH?

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- Yes, because pressure/density are related by an artificial equation of state (linear or nonlinear), thus the link to the Laplacian-type term in density is broadly equivalent to the compressible energy equation
- This implies any δ-SPH implementation can be put in a pseudo-time loop to give exact incompressibility (which was known already)
- Doing this maps to using artificial compressibility with inclusion of the Laplacian term
- At discrete/code level, exact equivalence doesn't hold, of course

$$\frac{Dp}{Dt} = -\gamma p \nabla \cdot u + \frac{\gamma v}{P_r} \nabla \cdot \nabla p$$

$$\frac{Dp^{i+1,k+1}}{D\tau} = -k_1 \nabla \cdot u^{i+1,k} - k_2 \nabla \cdot \left(\frac{Du^{i+1,k}}{Dt} + \frac{\nabla p^{i+1,k}}{\rho}\right) + \dots + \psi_{ij} = 2(\rho_j - \rho_i) \frac{\mathbf{r}_{ji}}{|\mathbf{r}_{ij}|^2} - \left[\langle \nabla \rho \rangle_i^L + \langle \nabla \rho \rangle_j^L\right]$$

$$\frac{D\rho_i}{Dt} = -\rho_i \sum_j (\mathbf{u}_j - \mathbf{u}_i) \cdot \nabla_i W(\mathbf{r}_j) V_j + \delta h c_0 \sum_j \psi_{ij} \cdot \nabla_i W(\mathbf{r}_j) V_j$$

$$p = c_0^2 (\rho - \rho_0)$$
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Case 1: Hydrostatic/elastic column

 10^{0}

€ (ⁿ(2)) € (ⁿ(2)) (ⁿ(2))) (ⁿ(2)) (ⁿ(2))) (ⁿ

 10^{2}

 $env(\delta_y) \ [\mu m]$

 10^{-1}

10⁻² L

- Hydrostatic column of water is released onto an elastic plate, recording structural displacements
- Acoustic waves in WCSPH interfere with structural response, adding complex amplitude and frequency components into coupled response
- Frequency response dominated by multiples of c_0 , dominant component at $\sim 0.5c_0/2H$



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Case2: Rotation of a square patch

- Large deformations of the free surface with negative pressures.
- Strongly dependent on particle remeshing or TIC approaches to avoid fragmentation.
- Very similar kinematics of the flow in AC and δ-SPH.
- Notably improved pressures recorded in the ACSPH scheme and apparent improvements in conservation properties.
- Smooth pressure even at coarsest resolution.





Axes normalised by time to account for spin-out

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Case 3: Dam break

- Iterative nature of ACSPH scheme ensures pressure boundary conditions are resolved on the first iteration, whereas WCSPH resolves this in time
- Initial fluid impact shows similar pressure profiles between AC and WCSPH, with reduced noise in ACSPH; subsequent fluid interaction induces strong pressure oscillations in WCSPH from acoustic waves







Case 3: Dam break (Cost)

- Requirement for iteration in ACSPH implies a higher cost?
- This is a balance between the stability criterion
 - WCSPH $\Delta T \propto h/c_0$
 - ACSPH $\Delta T \propto h$

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- And the number of iterations required to solve the scheme
- With correct parameterisation of the time-integration scheme, ACSPH can be as fast as δ -SPH.







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ACSPH 'works' harder where it is required

 \mathcal{C}^{e} (right): 'Number of function evaluations required to complete the solution'. Removes question of software optimisation and hardware

$$\mathcal{C}^e = \int_0^t \frac{m_{iter} s_{RK}}{\Delta t} dt$$

 \mathcal{C}^{w} (left): Wall time (on 1 P100 GPU)

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Case 4: Vertical sloshing

- Purely vertical excitation at 10.05 Hz and up to 7g acceleration
- WCSPH shows acoustic noise arising immediately from impulsive release and excitation
- This can be filtered out to recover the incompressible solution (as shown here)
- Filtering can not be applied in FSI cases; noise remains in structural accelerations –
- Structures (with inertia) act as a lowpass filter, reducing the high-frequency noise here



t/T1

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Conclusions

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- Weak compressibility can be problematic for some FSI cases. Acoustic effects appear in the calculated pressures/forces, and can manifest themselves in structural time histories. Forces from forced motion are hard to compare due to noise
- For our work, it was beneficial to move to full incompressibility, which was achieved with ACSPH. Or, this, can of course also be thought of as a pseudo-loop for the ISPH system, because...
- ...ISPH, ACSPH and δ-SPH are very closely associated. There is even no obvious reason to prevent mixing them, if it were to be beneficial
- This means AC ideas can be mapped to δ-SPH and ISPH
- It may be possible to construct other pressure smoothing terms, as tested in the JST context here